# Ch4: Automatic Controllers

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## Introduction



#### **Automatic Controllers**

An **Automatic Controller** compares the actual value of the plant output with the reference input, determines the deviation, and produces a control signal that will reduce the deviation to zero or to a small value with a desired transient response.





#### **Classifications of Common Controllers**

Most controllers may be classified according to their control actions (u) as:

- 1. On-Off Controller (Two-Position or Binary Controller)
- 2. Proportional (P) Controller
- 3. Integral (I) Controller
- 4. Derivative (D) Controller
- 5. PD, PI, and PID Controllers





#### **On-Off Controller**

**On-Off Controller (Two-Position** or **Binary** Controller): It has only two possible values (usually on and off) at its output u(t), depending on the input e(t) to the controller.

- This control system is relatively simple and inexpensive and is used very widely.
- Output oscillation between two limits is a typical response characteristic of this control system.





# **PID Controllers**



#### **Proportional (P) Controller**

**Proportional (P) Controller**: Output u(t) is **proportional** to its input e(t).

$$u(t) = K_P e(t) \longrightarrow U(s) = K_P E(s)$$
  
proportional gain

This controller is essentially an **amplifier** with an adjustable gain.





### Integral (I) Controller

**Integral (I) Controller**: Output u(t) is **proportional** to the **integral** of its input e(t).

$$u(t) = K_I \int_0^t e(\tau) d\tau \longrightarrow U(s) = \frac{K_I}{s} E(s) \longrightarrow K_I/s \longrightarrow U(s)$$
ntegral gain

In this controller, the control action u(t) at any instant is the area under the error signal curve up to that instant. Therefore, the control action is based on the **history** of the system error.



Note: The control action u(t) can have a nonzero value when the error signal e(t) is zero, but this is impossible in a Proportional controller. Thus, constant **disturbances** can be canceled with zero error.



### **Derivative (D) Controller**

**Derivative (D) Controller**: Output u(t) is **proportional** to the **derivative** of its input e(t).

$$u(t) = K_D \frac{de(t)}{dt} \longrightarrow U(s) = K_D s E(s) \longrightarrow K_D s U(s)$$
derivative gain

- Derivative control is essentially **anticipatory**, measures the instantaneous error, and predicts the large overshoot ahead of time.
- This controller tends to **increase** the **stability** and **sensitivity** of the system. However, it also tends to **amplify noise**.
- This controller does not affect the steady-state error directly, but it adds damping to the system and by increasing the gains the steady-state may be improved.
- It is always used in combination with Proportional (P) or Proportional-plus-Integral (PI) control action, i.e., PD or PID.



#### PD, PI, and PID Controllers

**PD, PI, and PID Controllers**: These controllers are **combinations** of proportional (P), derivative (D), and integral (I) controllers.

For example, PID Controller:  

$$u(t) = K_{P}e(t) + K_{I} \int e(t) dt + K_{D} \frac{de(t)}{dt} \longrightarrow \begin{cases} U(s) = \left(K_{P} + \frac{K_{I}}{s} + K_{D}s\right)E(s) \\ \text{or} \\ U(s) = K_{P}\left(1 + \frac{1}{T_{I}s} + T_{D}s\right)E(s) \end{cases}$$

$$\stackrel{R(s)}{\longrightarrow} \stackrel{E(s)}{\longrightarrow} \stackrel{PID}{\longrightarrow} \stackrel{G(s)}{\longrightarrow} \stackrel{C(s)}{\longrightarrow} \qquad \text{integral time} \qquad \text{derivative time}$$

- This controller has the advantages of each of the three individual control actions.
- If the system is second order or higher the use of PID controller is required if we wish to have arbitrary transient-response behavior and acceptable steady-state behavior.



#### Parameters of Step Response of Underdamped **Second-Order Systems**

The parameters defined for the step input response of underdamped second-order systems:

**1**. **Peak Time**  $T_n$ : The time required for the response to reach the **first** (or maximum) peak.

**2**. Maximum Overshoot  $M_p$ : The percentage of the steady-state value that the response overshoots the steady-state value at the peak time  $T_p$ .  $M_p = \frac{c(T_p) - c(\infty)}{c(\infty)} \times 100$ 

**3**. Settling Time T<sub>s</sub>: The time required for the response to reach and stay within 2% (or 5%) of the steady-state value.

**4**. **Rise Time**  $T_r$ : The time required for the response to go from 0% to 100% (or 10% to 90% or 5% to 95%) of the steady-state value.



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# Physical Realization of Controllers/Compensators



#### **Active-Circuit Realization**



**Note**: This transfer function contains a minus sign. Another circuit can be connected to either the input or the output of the circuit to act as a **sign inverter** as well as a gain adjuster.

 $Z_2(s)$ 





#### **Active-Circuit Realization: PID Controllers**

Function	$Z_1(s)$	$Z_2(s)$	$-rac{Z_2(s)}{Z_1(s)}$
P Controller	-	-	$-\frac{R_2}{R_1}$
I Controller	-		$-\frac{\frac{1}{RC}}{s}$
D Controller		-	-RCs
PI controller	-	-	$-\frac{R_2}{R_1} \frac{\left(s + \frac{1}{R_2C}\right)}{s}$
PD controller	$- \begin{bmatrix} C \\ \vdots \\ R_1 \end{bmatrix} - \begin{bmatrix} R_1 \\ \vdots \\ N \end{bmatrix} - \begin{bmatrix} R_1 \\ $	-	$-R_2C\left(s+\frac{1}{R_1C}\right)$
PID controller	$\begin{array}{c} C_1 \\ \hline \\ R_1 \\ \hline \\ \end{array}$	$\xrightarrow{R_2} C_2$	$-\left[\left(\frac{R_2}{R_1} + \frac{C_1}{C_2}\right) + R_2C_1s + \frac{1}{\frac{R_1C_2}{s}}\right]$



#### Example

Implement the following PID controller.

$$G_c(s) = \frac{(s+55.92)(s+0.5)}{s}$$



#### Answer:

Since there are four unknowns and three equations, we arbitrarily select a practical value for one of the elements, e.g.,  $C_2 = 0.1 \mu$ F, and find the remaining values.





#### **Active-Circuit Realization: Lag/Lead Compensators**





#### Passive-Circuit Realization: Lag/Lead Compensators

Lag, lead, and lag-lead compensators can also be implemented with passive networks.

